

FY.51421US1NP

PATENT

Specification

SHIFT CONTROL DEVICE FOR STRADDLE-TYPE VEHICLE, AND STRADDLE-TYPE VEHICLE

CROSS-REFERENCE TO RELATED APPLICATIONS

[10001] This application is a U.S. National Phase of International Application No. PCT/JP2005/011803, filed June 28, 2005, which claims priority to Japanese Application No. 2004-195632, filed July 1, 2004, each of which is hereby incorporated by reference in its entirety.

BACKGROUND OF THE INVENTION

Field of the Invention

[0002] ————— [0001]

—The present invention relates generally shift control devices for straddle-type vehicles, and more specifically, to a shift control device for a straddle-type vehicle for that is operative to electrically controlling a transmission of the straddle-type vehicle to change speeds, and to a straddle-type vehicle.

Description of the Related Art

[0003] ————— [0002]

— In some electric shift control devices, a conventional foot-operated shift pedal is not used, but an electric motor (shift actuator) is actuated based on a speed change command signal that is output from a shift switch to rotate the shift shaft of a transmission for shift change.

[0004] ————— [0003]

— In the case of shift change using a foot-operated shift pedal, repeated shift operations may be required to complete the shift change if when a dog in the transmission does not cannot be disengaged or engaged smoothly, repeated shift operations can eventually

complete the shift change. However, with an electric shift control device, the shift change might not be made when if a dog cannot be does not disengaged or engaged smoothly, smooth shift change cannot occasionally be made.

[0005] [0004]

In an attempt to address such a this problem related to electric shift control devices, a feedback method has been proposed. According to this method, to feed back the angle of a shift cam is detected and fed back in order to adjust the operation angle of the shift actuator for ensuring that the dog properly and smoothly disengages and engages smoothly. Although beneficial, this method can be problematic due to has the problem of slow shift speed and the complexity of the device.

[0006] [0005]

To This method is also problematic because in order to operate the shift actuator to-at a predetermined angle in a predetermined period, the shift actuator must keeps operating even during abutment of the dog. Due to the abutment with the shift actuator, and hence it is not possible to prevent the dog may tend to rotatefrom rotating with the operation of the shift actuator. It-Although it is possible to prevent the dog from rotating with the operation of the shift actuator, this requires the by, for example, interposition of an actuation force transmission mechanism, such as a spring mechanism between the shift actuator and the shift shaft. HoweverFurther, if the load required to disengage the dog cannot be obtained with the spring, the dog cannot be disengaged. In addition, if the stroke amount of the shift actuator needs to be increased, and the shift speed is made slower.

[0007] [0006]

In view of the foregoing issues, Japanese Patent Document No. JP-B-3044498 Patent Document+ discloses a technique for providing a lost motion mechanism constituted of an elastic member between the electric motor and the shift shaft. This lost motion mechanism is interposed between a speed reduction gear mechanism, (which is provided positioned between the output shaft and the shift drum shaft of the electric motor,) and the shift shaft in order to prevent the electric motor from being overloaded. After-Thus, instead of being applied to the shift actuator, any overload is applied to the elastic member and results in elastic deformation of the elastic member is overloaded and hence elastically

deformed. Therefore, when the shift shaft is rotationally driven by the resilient force, the shift shaft can be rotationally driven smoothly, without the influence of the inertial mass of the speed reduction gear mechanism, which allows. Such a configuration tends to ensure smooth speed change shift operation.

[0008] [0007]

Incidentally, albeit unrelated to electric shift control devices, Japanese Patent Document No. JP-Y-Sho 43-11555 Patent Document 2 discloses a technique for achieving smooth shift change using a foot-operated shift pedal, though not related to an electric shift control device. Specifically, This reference teaches a coupling mechanism that is disconnected at a portion between the shift pedal and the shift shaft. Both the disconnected ends of the coupling mechanism are linked via an elastic member and have with play equivalent to half the stroke of the shift pedal. With this structure, the dog can be disengaged with operation force of the shift pedal directly applied thereto, and can also be engaged always by the elastic force of the elastic member. This configuration tends to ensure which allows smooth shift change for foot-operated shift pedals.

[0009] Despite the beneficial shift control devices described in Japanese Patent Document Nos. JP-B-3044498 and JP-Y-Sho 43-11555, the elastic members are disposed in the engine case, for example, in the case of the elastic member of JP-B-3044498, between a reduced speed output gear, which is located at the final speed reduction end of the speed reduction gear mechanism, and the shift shaft. In the case of JP-Y-Sho 43-11555, the elastic member is disposed between a pedal shaft and a change arm. Hence, an existing structure cannot be utilized and high maintenance may be required. Therefore, there is a need in the art for an actuation force transmission mechanism that allows smooth shift change and is compactly sized in order to mitigate any restriction on installation location and enable easy installation.

Patent Document 1: JP-B-3044498

Patent Document 2: JP-Y-Sho 43-11555

SUMMARY OF THE INVENTION

[0010]

Disclosure of the Invention

Problem to be Solved by the Invention

[0008]

Conventional elastic members, however, are disposed in the engine case, for example, in the case of Patent Document 1, between a reduced speed output gear, which is located at the final speed reduction end of the speed reduction gear mechanism, and the shift shaft, and in the case of Patent Document 2, between a pedal shaft and a change arm, and hence an existing structure cannot be utilized and high maintenance, etc. is required.

The present invention has been made in view of the foregoing, and therefore has an object to provide an easy to maintain shift control device for a straddle-type vehicle allowing smooth shift change and utilizing an existing structure.

Means for Solving the Problem

As described herein, embodiments several aspects of the present invention relate to provides a shift control device for a straddle-type vehicle for performing shift control. In accordance with various implementations, in which a shift actuator is can be stroked by a predetermined amount to rotate a shift shaft, and a dog is can be engaged and disengaged by the rotation of the shift shaft. The embodiments described of the shift control device herein can thus provide for a smooth-shifting straddle-type vehicle incorporating the actuation force transmission mechanism.

[0011] The shift control device can includes: a transmission mechanism comprising, a biasing mechanism urging means, and a stopper mechanism. The transmission mechanism can also including: a first coupling part and a second coupling part coupled for movement relative to each other: an. The urging means biasing mechanism can be configured for urging the first and second coupling parts toward a neutral position; and a. The stopper mechanism can be configured for stopping the relative movement of the first or second coupling part when the first or second coupling part is moved relatively from the neutral position against urging force of the urging means biasing mechanism. The transmission mechanism is can be disposed outside an engine case and be interposed between the shift actuator and the shift shaft.

[0012] — In a preferred embodiment, the transmission mechanism is can be arranged such that, when resistive force acts against movement of the transmission mechanism, one of the first or and second coupling part moves relatively against the urging force of the urging meansbiasing mechanism until the respective first or second coupling part is stopped by the stopper mechanism. Further, and then the first and second coupling parts can then move together.

[0013] — In a preferred yet another embodiment, the urging meansbiasing mechanism can includes a first urging meansbiasing member disposed in the first coupling part and a second urging meansbiasing member disposed in the second coupling part. The, and the stopper mechanism can includes a first stopper mechanism for stopping relative movement of the first coupling part and a second stopper mechanism for stopping relative movement of the second coupling part.

[0014] — Preferably, the urging force of the first urging meansbiasing member and the urging force of the second biasing member urging means are can be equal to each other.

[0015] — In a preferred accordance with another embodiment, the first and second coupling parts are can be coupled for movement relative to each other in sliding directions (e.g., linearly).

[0016] — In a accordance with yet another preferred embodiment, the urging meansbiasing mechanism can includes a compression spring.

[0017] — In accordance with yet another a preferred embodiment, the first and second coupling parts are can be coupled for movement relative to each other in rotating directions.

[0018] — In a preferred accordance with yet another embodiment, the urging meansbiasing mechanism can includes a leaf-type pine needle-like spring. In a preferred embodiment, the leaf spring has an elongated rod-like shape (-that is, configured substantially as a needle).

[0019] — In a preferred embodiment accordance with yet another embodiment, the transmission mechanism is can be disposed on the shift shaft.

[0020] — In accordance with yet another embodiment a preferred embodiment,

the transmission mechanism ~~is~~can be disposed on a rotation axis of a gear of a speed reduction mechanism coupled to the shift actuator.

[0021] — In accordance with yet another embodiment ~~a preferred embodiment~~, the shift actuator ~~is~~can be coupled to the shift shaft via a coupling mechanism for transmitting actuation force of the shift actuator. ~~In addition,~~ and the transmission mechanism ~~is~~can be held by the coupling mechanism.

[0022] — In accordance with yet another embodiment ~~a preferred embodiment~~, the transmission mechanism ~~is~~can be provided in a case held by the coupling mechanism.

[0023] — In accordance with yet another embodiment ~~a preferred embodiment~~, the shift actuator ~~is~~can be coupled to the shift shaft via a coupling mechanism for transmitting actuation force of the shift actuator; and the coupling mechanism is of adjustable length.

— The present invention provides a straddle-type vehicle incorporating the shift control device constructed as described above.

Effect of the Invention

[0025] — Embodiments of the shift control device for a straddle-type vehicle ~~of the present invention~~ can therefore allows smooth shift change even when disengagement of the dog is difficult or dog abutment occurs during engagement of the dog.

[0024] — Finally, embodiments of the transmission mechanism described above ~~herein~~ can be disposed outside an engine case ~~that houses~~ ~~which includes~~ the shift shaft. In this way, the transmission mechanism can be provided without the need to modify the inside of the engine case and can be easily maintained.

BRIEF DESCRIPTION OF THE DRAWINGS

[0025] — FIGs. Figures 1(a) and 1(b) are conceptual schematic diagrams showing the basic structure of a shift control device for a straddle-type vehicle, which includes an actuation force transmission mechanism, configured according to an embodiments of the present invention.

[0026] — Figures FIGs. 2(a) to 2(e) show how ~~a~~ the transmission mechanism ~~10~~ of

Figure 1(a) can operates when a shift actuator is stroked by a predetermined amount in accordance with an implementation of the present invention.

[0027] Figures FIGs. 3(a) to 3(g) show a specific structure and operation of the actuation force transmission mechanism +0-in accordance with an embodiment the present invention.

[0028] FIG. Figure 4 is a graph showing the rotational angle of a shift shaft versus the stroke length of the shift actuator in accordance with an embodiment of the present invention.

[0029] FIG. Figure 5 shows how a neutral position is-can be set using coil springs of different urging forces in accordance with an embodiment of the present invention.

[0030] FIG. Figure 6 is a side view of a two-wheeled motor vehicle in accordance with an embodiment of the present invention.

[0031] FIG. Figure 7 is a plan view of an engine provided with the shift actuator, etc.,-in accordance with an embodiment of the present invention.

[0032] FIG. Figure 8 is a side view of the engine provided with the shift actuator, etc., in accordance with an embodiment of the present invention.

[0033] FIG. Figure 9 is an exploded perspective view of a transmission mechanism in accordance with an embodiment of the present invention in the present invention.

[0034] FIG. Figure 10 shows the developed shape of grooves in a shift cam in accordance with an embodiment of the present invention.

[0035] FIG. Figure 11 is a side view of the shift actuator, etc., in accordance with an embodiment of the present invention.

[0036] FIG. Figure 12 is a perspective view of an actuation force transmission mechanism in accordance with an embodiment of the present invention.

[0037] FIG. Figure 13 is another perspective view of the actuation force transmission mechanism in accordance with an embodiment of the present invention, viewed from a direction different from that illustrated in FIG. Figure 12.

[0038] Figure FIG. 14 is a front view of the actuation force transmission mechanism in accordance with an embodiment of the present invention, viewed from the direction of the arrow A in FIG. Figure 12.

[0039] FIG. Figure 15 is a right side view corresponding to FIG. Figure 14.

[0040] FIG. Figure 16 is a plan view corresponding to FIG. Figure 14.

[0041] FIG. Figure 17 is a block diagram showing an engine control unit, etc., in accordance with an embodiment of the present invention.

[0042] FIG. Figure 18 shows an actuation force transmission mechanism according to yet another embodiment of the present invention in a normal state, in which FIG. Figure 18(a) is a plan view of an embodiment of the actuation force transmission mechanism, FIG. Figure 18(b) is a sectional view taken along the line B-B of FIG. Figure 18(a), and FIG. Figure 18(c) is a sectional view taken along the line C-C of FIG. Figure 18(a).

[0043] FIG. Figure 19 shows the actuation force transmission mechanism according to the another embodiment of the present invention in the shortened state, in which FIG. Figure 19(a) is a plan view of the actuation force transmission mechanism, and FIG. Figure 19(b) is a sectional view corresponding to FIG. Figure 19(a).

[0044] FIG. Figure 20 shows the actuation force transmission mechanism according to the another embodiment of the present invention in the longest expanded state, in which FIG. Figure 20(a) is a plan view of the actuation force transmission mechanism, and FIG. Figure 20(b) is a sectional view corresponding to FIG. Figure 20(a).

[0045] FIG. Figure 21 shows the actuation force transmission mechanism according to the another embodiment of the present invention in a divided state.

[0046] FIG. Figure 22 shows the structure of a stopper member in another embodiment of the present invention.

[0047] FIG. Figures 23(a) and 23(b) show the structure of the stopper member in still another embodiment of the present invention.

~~Description of Reference Numerals and Symbols~~

~~[0027]~~

~~10: transmission mechanism~~

~~11a: first coupling part~~

~~11b: second coupling part~~

12: urging means
12a: first urging means (coil spring)
12b: second urging means (coil spring)
13: stopper mechanism
13a: first stopper mechanism (stopper member)
13b: second stopper mechanism (stopper member)
15: support member
16a: first opening
16b: second opening
151: engine
152: engine case
155: speed change mechanism
156: shift fork
157: slide rod
158: shift cam
159: shift shaft
160: ratchet mechanism
161: shift arm
162: stopper plate
164: actuation force transmission mechanism
165: shift actuator
166: pinion gear
167: coupling rod
170: rotary frame (first coupling part)
171: pine needle like spring (urging means)
172: support bar
174: fixed lever (second coupling part)
177: actuation force transmission mechanism
179: first coupling part
180: second coupling part

181: coil spring (urging means)

Best Mode for Carrying Out the Invention

[0028]

DETAILED DESCRIPTION OF THE PREFERRED EMBODIMENT

[0048] — Referring With reference now to the drawings, wherein the figures are provided for purposes of illustrating preferred embodiments of the present invention and not for purposes of limiting the same, FIGS. 1-3 illustrate embodiments of Before starting the description of specific structures of a shift control device for a straddle-type vehicle of the present invention, the basic concept of the present invention is first described with reference to FIGs. 1 to 3. A basic description of the actuation force transmission mechanism will be provided first, followed by a detailed description of specific structures utilizable in accordance with embodiments of the present invention.

[0049] — FIGs FIGS. 1(a) and 1(b) are conceptual diagrams showing the basic structure of an embodiment of the shift control device for a straddle-type vehicle of the present invention.

[0050] — [0029]

— FIG. 1(a) is a conceptual diagram showing a transmission mechanism 10 interposed between a shift actuator and a shift shaft in the shift control device in accordance with an embodiment of the present invention. Normally, the shift actuator is can be coupled to the shift shaft through a coupling rod or the like. The shift actuator is can be stroked by a predetermined amount to rotate the shift shaft. The rotation of the shift shaft can engages and disengages a dog to control shift change. In an embodiment, The transmission mechanism 10 is can be disposed at an intermediate portion of the coupling rod.

[0051] — [0030]

— As shown in the embodiment illustrated in FIG. 1(a), the transmission mechanism 10 can includes a first coupling part 11a and a second coupling part 11b coupled for movement relative to each other. The transmission mechanism 10 can also include, an urging means a biasing mechanism 12 for urging the first and second coupling parts 11a, 11b toward a neutral position. Finally, the transmission mechanism 10 can also include, and a stopper mechanism 13 for stopping relative movement of the first or second coupling part

11a, 11b when they move relative to each other from the neutral position against the an urging force of the urging meansbiassing mechanism 12.

[0052] [0031]

Another embodiment of the transmission mechanism 10 shown in FIG. 1(b) has a structure similar to that shown in FIG. 1(a), but is provided with an urging meansbiassing mechanism 12 and a stopper mechanism 13 for each of the first and second coupling parts 11a, 11b. Thus, the first coupling part 11a is can be provided with a first urging meansbiassing member 12a and a first stopper mechanism 13a for stopping relative movement of the first coupling part 11a. In like manner, while the second coupling part 11b is can be provided with a second urging meansbiassing member 12b and a second stopper mechanism 13b for stopping relative movement of the second coupling part 11b. As discussed below, the urging meansbiassing mechanism can be a resilient component variously sized and configured to assist in the return the first and second coupling parts 11a, 11b to or from the neutral position. Further, the stopper means can also be variously sized and configured to assist in limiting the movement of the first and second coupling parts 11a, 11b. The operation of the actuation force transmission mechanism 10 shown in FIG. 1(b) is similar to that of the actuation force transmission mechanism 10 shown in FIG. 1(a), and hence only the latter is described here.

[0053] [0032]

According to implementations of the present inventionNow, the operation of the above transmission mechanism 10 is now described with reference to FIGsFIGS. 2(a) to 2(e). The operation of the transmission mechanism 10 shown in FIG. 1(b) is basically the same as that of the transmission mechanism 10 shown in FIG. 1(a), and hence only the latter is described here.

[0054] [0033]

FIGsFIGS. 2(a) to 2(e) show how the transmission mechanism 10 can operates when the shift actuator is stroked by a predetermined amount.

[0055] [0034]

FIG. 2(a) shows a state in which the first coupling part 11a and the second coupling part 11b are held at the neutral position of the transmission mechanism 10 by the

urging force of the urging meansbiassing mechanism 12. After the shift actuator is stroked by a predetermined amount and a shift up or a shift down is completed, the shift actuator can returns to a predetermined position. If the first and second coupling parts 11a, 11b deviate from the neutral position ~~is deviated~~ at that time, however, the dog ~~is~~can become disengaged and may subsequently be engaged at deviated positions by the rotation of the shift shaft at the next shift up or shift down. This condition, ~~which~~ may hinder smooth shift change. ~~Thus~~However, the urging force of the urging meansbiassing mechanism 12 needscan be to be preset such that the first and second coupling parts 11a, 11b neutral position ~~will~~can be prevented from deviating from the neutral position.

[0056] ————— [0035]

Referring ~~With reference~~ still to FIGS. 2(a)-(e), ~~w~~When the shift actuator in this state is actuated ~~based on~~in response to a gear change command signal and starts being stroked by a predetermined amount, an actuation force F1 in the direction of the arrow (labeled F1) ~~is~~can be applied to the transmission mechanism 10 from the shift actuator side (the right side of the drawing) as shown in FIG. 2(a). At this time, when some resistive force R1 (which will be described specifically later) acts against movement of the transmission mechanism 10 on the shift shaft side ~~(the left side of the drawing)~~ of the transmission mechanism 10, the urging meansbiassing mechanism 12 (e.g. a compression spring) ~~is~~can be compressed, and as a result, the first coupling part 11a can moves relatively from the ~~a~~ central neutral position, as shown in FIG. 2(ba), to a position shown in FIG. 2(b). ~~The~~ As also shown, the first coupling part 11a can moves relatively against the urging meansbiassing mechanism 12 until ~~it~~ the movement of the first coupling part 11a is stopped by action of the stopper mechanism 13, as shown in FIG. 2(b).

[0057] ————— [0036]

When the relative movement of the first coupling part 11a relative to the second coupling part 11b is stopped, ~~then~~ the first coupling part 11a and the second coupling part 11b can move together as shown in FIGS. 2(b)-(c). At this time, the transmission mechanism 10 can moves in ~~as it were~~ a “rigid” state and ~~hence~~ can therefore be enabled to move against the resistive force R1 to effectively rotate the shift shaft.

[0058] ————— [0037]

When the resistive force R1 is no longer applied against the movement of the transmission mechanism 10, as shown in FIG. 2(d), the urging force of the urging meansbiasing mechanism 12 can urges the first coupling part 11a toward the neutral position, and the transmission mechanism 10 can keeps moving as the shift actuator is stroked.

[0059] [0038]

Then, when some resistive force R2 (which will be described specifically later) acts against the movement of the transmission mechanism 10 again, the urging meansbiasing mechanism 12 is can be compressed as shown in FIG. 2(e), and as a result, the first coupling part 11a can moves relatively against the urging meansbiasing mechanism 12 to a point before it is stopped by the stopper mechanism 13 in the same way as in FIG. 2(b). When the relative movement of the first coupling part 11a is stopped, the second coupling part 11b is can be urged by the urging meansbiasing mechanism 12 against the resistive force R2. Without the resistive force R2, the second coupling part 11b is can be moved by the urging force of the urging meansbiasing mechanism 12.

[0060] [0039]

As described above, when some resistive force acts against movement of the transmission mechanism 10 in which the first coupling part 11a and the second coupling part 11b are coupled to each other, the urging meansbiasing mechanism 12 and the stopper mechanism 13 can work in conjunction with each other to relatively move the first coupling part 11a (or the second coupling part 11b) for a certain period in order to relieve the resistive force. After the certain period, the first coupling part 11a and the second coupling part 11b can move together to allow the actuation force of the shift actuator to act directly on the shift shaft.

[0061] [0040]

The above description describes a typical example of the operation of the transmission mechanism 10. The operation of the transmission mechanism 10 may can vary depending on the magnitude and duration of resistive force which acts on the transmission mechanism 10, the stroke length of the shift actuator, etc.

[0062] [0041]

For example, in the case where the resistive force R1 is applied to the

transmission mechanism 10 of the above example for only a short period, the compression of the urging meansbiassing mechanism 12 may not move the first coupling part 11a far enough relatively to the second coupling part 11b to cause the first coupling part 11a to be before it is stopped by the stopper mechanism 13. Instead, but may allow the first coupling part 11a can to return toward the neutral position when the resistive force R1 is no longer applied.

[0063] ————— [0042]

————— In the case where the shift actuator is stroked in the opposite direction, the transmission mechanism 10 can basically performs the same operation as shown in FIGsFIGS. 2(a) to 2(e). In such as case, since the transmission mechanism 10 hascould have a targetsymmetrical structure with respect to the neutral position.

[0064] ————— [0043]

————— In the operation of the transmission mechanism 10 of the above example, the first coupling part 11a and the second coupling part 11b arecan be coupled so as to be movable relative to each other in sliding directions. However, the first coupling part 11a and the second coupling part 11b maycan also be coupled so as to be movable relative to each other in rotating directions.

[0065] ————— [0044]

————— The foregoing describes the exemplary conceptual structures and operations of the embodiments of the transmission mechanism 10. Now, exemplary a specific structures and operations of embodiments of the transmission mechanism 10 are described in association with actual engagement and disengagement of the dog with reference to FIGsFIGS. 3 and 4.

[0066] ————— [0045]

————— FIGsFIGS. 3(a) to 3(g) show exemplary the operation of an embodiment of the transmission mechanism 10 and the exemplary operation of an embodiment of a dog mechanism. FIG. 4 shows the rotational angle of the shift shaft versus the stroke length of the shift actuator, according to an implementation of the present invention. The According to one embodiment, the transmission mechanism 10 described here has can have first and second coupling parts that each have an urging mechanism and a stopper mechanism separately for the first and second coupling parts. However, its basic operation is the same as

a transmission mechanism with one ~~urging means~~biassing mechanism and one stopper mechanism.

[0067] _____ [0046]

_____ The right side of FIG. 3(a) shows an embodiment of the transmission mechanism 10 with the first coupling part 11a and the second coupling part 11b held in the neutral position. ~~The, and~~ the left side of FIG. 3(a) shows an embodiment of the dog mechanism with a dog 20 engaged with a gear 21.

[0068] As shown in FIG. 3(a). _____ [0047]

_____ ~~T~~he first coupling part 11a of the transmission mechanism 10 ~~is~~can be inserted into an opening of, and thus slideably coupled to the second coupling part 11b. A first coil spring 12a can act as a ~~urging means~~biassing member, ~~and along with~~ and a first stopper member 13a, can be are disposed in an opening 16a of the first coupling part 11a. Likewise, a second coil spring 12b can act as an ~~urging means~~biassing member, ~~and along with~~ a second stopper member 13b, can be are disposed in an opening 16b of the second coupling part 11b.

[0069] _____ [0048]

_____ When a gear change command signal is input to the shift actuator in this state, the shift actuator can subsequently be ~~starts being~~ stroked by a predetermined amount. ~~The Referring~~ With reference now to FIG. 4, the shift shaft normally has “play” and can thus rotates by the play when the shift actuator is first stroked (represented by the diagonal line on the graph intermediate numbers 1 to ~~and~~ 2 on the horizontal axis of FIG. 4).

[0070] _____ [0049]

_____ As the shift actuator is further stroked, disengagement of the dog can starts. ~~Since~~ The frictional force of the dog 20 in engagement with the gear 21 can acts as resistive force against the movement of the shift actuator as shown in FIG. 3(b). Thus, according to an implementation of the present invention, the transmission mechanism 10 interposed between the shift actuator and the shift shaft can operates in such a way that:- the first coil spring 12a provided disposed in the first coupling part 11a ~~is~~can become compressed. As and as a result, the second coupling part 11b can moves relatively from the central neutral position.

[0071] [0050]

The second coupling part 11b can moves relatively against the first coil spring 12a until the first stopper mechanism 13a comes in contact with the sidewall of a support member 15 of the second coupling part 11b. While the support member 15 abuts the first stopper mechanism 13a, the first coupling part 11a and the second coupling part 11b are in a “rigid” state; the shift shaft does not rotate as the shift actuator is stroked during this stage of stroke (represented by the horizontal line on the graph intermediate numbers number 2 to and 3 on the horizontal axis of FIG. 4).

[0072] [0051]

When Furthermore, when the relative movement of the second coupling part 11b is stopped, then the first coupling part 11a and the second coupling part 11b can move together as shown in FIG. 3(c). At this time, since the transmission mechanism 10 moves in as it were a “rigid” state, the actuation force of the shift actuator is applied directly to the shift shaft and exceeds the above-described frictional force so that the dog 20 disengages from the gear 21 during this stage of stroke (represented by the diagonal line on the graph intermediate numbers 3 to and 4 on the horizontal axis of FIG. 4).

[0073] [0052]

When the dog 20 is completely disengaged, frictional force of the dog 20 with gear 21 no longer exists. Thus, the urging force of the first coil spring 12a can then returns the second coupling part 11b toward the neutral position as shown in FIG. 3(d). After the dog 20 is disengaged, the shift shaft can rotates with almost no resistive force acting against the movement of the transmission mechanism 10 (represented by the diagonal line on the graph intermediate numbers number 4 to and 5 on the horizontal axis of FIG. 4).

[0074] [0053]

Then, as shown in FIG. 3(e), resistive force due to abutment of the dog acts against the movement of the shift actuator when the dog 20 engages with a gear 22. Again, as shown in FIG. 3(f), the first coil spring 12a provided disposed in the first coupling part 11a is can become compressed, and as a result the second coupling part 11b can then moves relatively from the central neutral position. In the abutment of the dog 20, small urging force of the first coil spring 12a acts on the dog 20, and allows the dog 20 to engage with the gear

22 smoothly (represented by the horizontal line on the graph intermediate numbers 5 to and 6 on the horizontal axis of FIG. 4). When Once the dog 20 is completely engaged with the gear 22, there no longer exists resistive force as shown in FIG. 3(g). Thus, the urging force of the first coil spring 12a can returns the second coupling part 11b toward the neutral position.

[0075] [0054]

Preferably, a gap may can be provided so that the second coupling part 11b will move relatively not to be stopped by the first stopper mechanism 13a when the shift actuator is fully stroked and in the abutment of the dog, as shown in FIG. 3(f).

[0076] [0055]

As described above, in an embodiment of the shift control device of the present invention described above, the transmission mechanism 10 can includeing a first coupling part 11a and a second coupling part 11b, and can be coupled for so as to provide movement relative to each other. Further, the transmission mechanism 10 can be is interposed between the shift actuator and the shift shaft. When the shift actuator is stroked by a predetermined amount, the dog is can be compulsorily disengaged as the first and second coupling parts 11a, 11b are moved together by means of the stopper mechanism 13 (13a, 13b). Further, the dog can be and engaged (in the abutment of the dog) as the one of the first or and second coupling part is 11a. 11b is moved relatively against the urging force of the urging meansbiasing mechanism 12 (13a12a, 13b12b). This can facilitate allows smooth shift change.

[0077] [0056]

In the above description, the dog is can be disengaged as the first and second coupling parts move together, such as when the frictional force of the dog is great. However, it should be understood that the dog can be successfully disengaged as one of the first or and second coupling parts moves relatively, such as when in the case where the frictional force of the dog is small.

[0078] [0057]

The According to an implementation, the transmission mechanism 10 described above has an independent structure and hence can be disposed outside an engine case which includes houses the shift shaft. In this way, the transmission mechanism 10 can

be provided without the need to modify the inside of the engine case and can be easily maintained.

[0079] ————— [0058]

———— In addition, the transmission mechanism 10 described above can be easily disposed outside the engine case. For example, when the transmission mechanism 10 is can be held by a coupling mechanism (a mechanism for transmitting actuation force of the shift actuator to the shift shaft; for example, a coupling rod, a speed reduction mechanism, etc.) that can be coupled to the shift actuator and the shift shaft. Further, the transmission mechanism 10 described above can be effectively protected from water and dust by disposing it in a case held by the coupling mechanism.

[0080] ————— [0059]

———— The shift actuator may can be coupled to the shift shaft via a coupling mechanism of adjustable length for transmitting actuation force of the shift actuator.

[0081] ————— [0060]

———— In the case where the urging forces of the first and second coil springs 12a, 12b provided in the first coupling part 11a and the second coupling part 11b are the same in the transmission mechanism 10 shown in FIG. 3, the neutral position can be easily set comparatively easily. However, in the case whereif the urging forces are intentionally different, the neutral position must should be set carefully. NowWith reference now to FIG. 5, description will be made of how the neutral position is can be set using coil springs 12a, 12b that have of different urging forces with reference to FIG. 5.

[0082] ————— [0061]

———— As shown in FIG. 5(a), the free length of the first coil spring 12a (spring constant: N1) provided in the first coupling part 11a is defined as L1, and the free length of the second coil spring 12b (spring constant: N2) provided in the second coupling part 11b is defined as L2. Assuming that the first coupling part 11a and the second coupling part 11b of FIG. 5(b) are in the neutral position, and also the lengths of the first coil spring 12a and the second coil spring 12b are respectively x and y, the following equations hold true:

$$x + y + a = z \quad (1)$$

$$N1 \times (L1 - x) = N2 \times (L2 - y) \quad (2)$$

[0083] — The length x of the first coil spring 12a and the length y of the second coil spring 12b can be determined by solving these simultaneous equations (1), (2).

[0084] — [0062]

— The basic structure of the shift control device for a straddle-type vehicle according to embodiments of the present invention has been described above. Hereinafter, specific structures and operations of various embodiments of the shift control device will be described in detail with reference to FIGsFIGS. 6 to 23.

[0085] — [0063]

— FIGsFIGS. 6 to 17 show a specific structure of the shift control device according to an embodiment of the present invention. In FIG. 6, reference numeral 140 denotes a two-wheeled motor vehicle as a “straddle-type vehicle”, which can be provided with a front wheel 141 on its front side, a rear wheel 142 on its rear side, a fuel tank 144 in rear of handlebars 143, a seat 145 in rear of the fuel tank 144, and an engine 151 supported by a body frame below the fuel tank 144 and the seat 145.

[0086] — [0064]

— A transmission (not shown) is can be disposed in an engine case 152 for the engine 151. The transmission has can have four to six speeds and adopts a dog clutch. Power from a crankshaft of the engine 151 is can be transmitted to a main axle, and then to a drive axle via gears and dogs for respective speeds.

[0087] — [0065]

— Speed change operation of the transmission is can be achieved by a speed change mechanism 155, such an embodiment of which is as shown in FIG. 9. As shown in FIG. 9, the speed change mechanism 155 can includes shift forks 156 for regularly moving slide gears of the transmission, slidably mounted on a slide rod 157, and a rotatable shift cam 158 for sliding the shift forks 156.

[0088] — [0066]

— Cam grooves 158a are can be formed on the periphery of the shift cam 158. When developed, the cam grooves 158a are can be formed as shown in the exemplary embodiment of FIG. 10. The shift forks 156 are can be adapted to slide along the cam grooves 158a.

[0089] ————— [0067]

———— The According to an embodiment, the shift cam 158 can rotates via a ratchet mechanism 160 as a shift shaft 159 rotates. The ratchet mechanism 160 can be configured to provide a ratchet function for both forward and reverse directions to change one gear at a time. For example, the ratchet mechanism 160 can rotates the shift cam 158 with constant intervals (such as by a constant angle) to move the shift forks 156 regularly, or in other words has a ratchet function for both forward and reverse directions to change one gear at a time. A shift arm 161 of the ratchet mechanism 160 transmits rotation of the shift shaft 159, and can also restricts the stroke of shift shaft 159 in order to prevent the shift cam 158 from overrunning. A stopper plate 162 of the ratchet mechanism 160 can be utilized to keeps the shift cam 158 in specified positions.

[0090] ————— [0068]

———— The shift shaft 159 can moves rotationally in a predetermined direction through a device such as described below.

[0091] ————— [0069]

———— Referring With reference now to the embodiment illustrated in FIG. 7, Aa distal end 159a of the shift shaft 159 can projects from the engine case 152 to the outside of the engine, and is-can be provided with an actuation force transmission mechanism 164. The shift shaft 159 is-can be rotated by driving force of the shift actuator 165 via the actuation force transmission mechanism 164.

[0092] ————— [0070]

———— As shown in FIGsFIGS. 7 and 8, the shift actuator 165 is-can be disposed on a side of the upper part of the engine case 152 along the longitudinal direction of the vehicle. As shown in FIG. 11, the shift actuator 165 is-can be provided with a worm-worm gear 165a at the distal end of its rotary shaft. The worm-worm gear 165a is-can be configured to meshed with a pinion gear 166. A coupling shaft 166a is-can be provided eccentrically with respect to the center axis of the pinion gear 166.

[0093] ————— [0071]

———— Referring As seen in -again to FIG. 7, One end 167a of a coupling rod 167 extending vertically is-can be coupled to the coupling shaft 166a for free rotation as shown in

FIG. 7. Additionally, while the another end 167b of the coupling rod 167 is can be coupled to the actuation force transmission (conveyance) mechanism 164 as shown in FIG. 8.

[0094] [0072]

According to another embodiment, the coupling rod 167 may can be constituted with two halves. One of the halves can, one having have a male-threaded end and the other halve can have having a female-threaded end. In this regard, and the two halves may can be configured to be coupled to each other by screwing and then fixing them with a nut. This structure can allows for the adjustment of the distance between the shift actuator 165 and the shift shaft 159 by loosening the nut, and rotating one half of the coupling rod 167 to adjust its length, and then tightening the nut again to complete the adjustment.

[0073]

[0191] As shown in an embodiments of FIGS FIGS. 12 to 16, in the actuation force transmission mechanism 164, a rotary frame 170, to which the other end 167b of the coupling rod 167 is can be coupled to a rotary frame 170. In this regard, the rotary frame 170 of the actuation force transmission mechanism 164, is can be disposed around the shift shaft 159 for free rotation relative to the shift shaft 159. The Additionally, the other end 167b of the coupling rod 167 is can be coupled to a coupling recess 170a of the rotary frame 170 for free rotation. The rotary frame 170 is can be provided with an actuation piece 170b that can be configured bent to project therefrom, such as by being bent from the rotary frame 170 or otherwise connected thereto. The actuation piece 170b is can be inserted between two support bars 172 of a pine needle like spring 171, which functions as a biasing mechanism. Each bar preferably has a thin elongated rod-like shape, similar to a pine needle. as an “urging means,” which can be configured substantially needle like. The two support bars 172 can urge the actuation piece 170b toward the neutral position shown in FIGS FIGS. 13 and 15.

[0095] [0074]

Also, in accordance with another implementation of the actuation force transmission mechanism 164 shown in FIGS. 12-16, a fixed lever 174 is can be fixed to the distal end 159a of the shift shaft 159. The As shown in FIGS. 13-16, the fixed lever 174 is can be provided with a pin-to-be-pressed 174a projecting therefrom. The pin-to-be-pressed 174a, and is can be inserted between the pair of support bars 172.

[0096] ————— [0075]

With this structure, when the rotary frame 170 is moved rotationally in an arbitrary direction from the neutral position, the actuation piece 170b can presses one of the two support bars 172 while the other of the support bars 172 can presses the pin-to-be-pressed 174a. In such a manner, so that the shift shaft 159 is can be moved rotationally in an arbitrary direction by a predetermined amount via at least due in part to the fixed lever 174. At this time In addition, the shift shaft 159 is can also be moved rotationally by the urging force of the support bars 172.

[0097] ————— [0076]

When Furthermore, the rotary frame 170 is further can moved rotationally rotate from that state against the urging force of the spring bars 171, 172 and to be moved relative to the fixed lever 174. Such movement of the rotary frame 170 relative to the fixed lever 174 can be by a predetermined amount in a rotational direction, and then the pin-to-be-pressed 174a of the fixed lever 174 is can be contacted and hence pressed by one of a pair of stopper edges 170c, 170c of the rotary frame 170. This contact can function as a “stopper means”. Thus, the relative movement of the rotary frame 170 and relative to the fixed lever 174 in a rotational direction is can be stopped. In this regard, and the rotational force of the rotary frame 170 can acts directly on the fixed lever 174, so that the shift shaft 159 moves rotationally together with the fixed lever 174.

[0098] ————— [0077]

Meanwhile, an engine control unit 210 for controlling the engine 151 is can be provided as shown in FIG. 17. In accordance with an implementation of the embodiment, various components can be connected to the engine control unit 210; such components can include are connected an engine speed sensor 211, a vehicle speed sensor 212, a clutch actuator position sensor (potentiometric sensor) 213, a shift actuator position sensor 214, a gear position sensor 215, an UP switch 216 for shifting up, and a DOWN switch 217 for shifting down. Detected values and operation signals from these components are can be input to the engine control unit 210. The In a preferred embodiment, the UP switch 216 and the DOWN switch 217 are can be provided on the handlebars 143.

[0099] ————— [0078]

The As also shown in FIG. 17, the engine control unit 210 is can be connected to a clutch actuator 218, the shift actuator 265165, a gear position display section 219, an engine ignition section 220, and a fuel injection device 221, which are can be driven and controlled based on the signals from the various sensors 211, etc.

[0100] [0079]

The signals from the UP switch 216, the DOWN switch 217, the shift actuator position sensor 214, the gear position sensor 215, etc., are can be input to the engine control unit 210, and control signals from the engine control unit 210 are can be used to drive and control the shift actuator 165.

[0101] [0080]

Next, the various functions of embodiments of the present invention will be described.

[0102] [0081]

To In order to change speeds of the transmission, the UP switch 216 or the DOWN switch 217 can be provided on the handlebars 143. These switches 216, 217 can be is operated to actuate the shift actuator 165 so as in order to rotate the worm gear 165a in a predetermined direction by a predetermined amount.

[0103] [0082]

Then, the pinion gear 166, shown illustratively in FIG. 11, can then rotates in a predetermined direction. In addition, the coupling shaft 166a, which can be disposed eccentrically with respect to the pinion gear 166, can moves rotationally, so that the coupling rod 167 is pushed downward or pulled upward.

[0104] [0083]

In addition, As the rotary frame 170 can moves rotationally in a predetermined direction via the coupling rod 167. This rotational movement can cause, the actuation piece 170b of the rotary frame 170 to presses one of the two support bars 172. In turn, This can causes the other of the support bars 172 to elastically press the pin-to-be-pressed 174a of the fixed lever 174. The pressing of the support bars 172 against the pin-to-be-pressed 174a can then, which in turn cause the moves rotational movement of the shift shaft 159 rotationally in a predetermined direction via the fixed lever 174.

[0105] ————— [0084]

————— When the shift shaft 159 is moved rotationally in this way, the shift cam 158 is can moved rotationally in a predetermined direction via the ratchet mechanism 160. Further, and the shift forks 156 are can then be guided by the cam grooves 158a to slide in predetermined directions. Thus, the slide gears of the transmission are can thus be moved, and the dog for a predetermined gear is can be disengaged while the dog for another is engaged.

[0106] ————— [0085]

————— When the dog is to be engaged, there are cases where the dog contacts another dog due to bad timing and hence is not engaged immediately. Even in such cases, the dogs are can be subjected to comparatively small urging force of the two support bars 172 and hence do may not abut against each other with large force. Thus, the components are can be protected from damage or the like. After that, the slide gears can move rotationally slightly, and the urging force of the rotational movement can causes the dogs to be meshed with each other reliably.

[0107] ————— [0086]

————— At According to an embodiment of the present invention, at the time when the two support bars 172 are elastically deformed and the rotary frame 170 and the fixed lever 174 are moved relatively in a rotational direction by a predetermined amount, one of the stopper edges 170c of the rotary frame 170 can contacts the pin-to-be-pressed 174a of the fixed lever 174. This can causes the rotary frame 170 and the fixed lever 174 to move rotationally together. Thus, even when the dog is engaged and difficult to be disengaged due to residual torque, the dog can be compulsorily disengaged.

[0108] ————— [0087]

————— To put it in other words, with a simple modification of the structure, the dog can be disengaged and engaged reliably and easily without precise control, even in the case where the shift operation is performed not manually, but mechanically using the shift actuator 165.

[0109] ————— [0088]

————— The According to another embodiment, the actuation force transmission

mechanism 164 is can be disposed on the axis of the shift shaft 159, as shown in FIG. 8, thereby achieving a compact structure. The actuation force transmission mechanism 164 can also be, which is disposed outside the engine 151, and can nevertheless be protected from water and dust easily by providing a case 192 for covering it.

[0110] [0089]

In accordance with another embodiment, the actuation force transmission mechanism 164 can be disposed on the axis of a gear shaft 190 of a damping mechanism 191, and be coupled to the shift actuator 165 as shown in FIG. 8. Such an embodiment can be an alternative to coupling the actuation force transmission mechanism 164, rather than on the axis of the shift shaft 159.

[0111] [0090]

Next, FIGS~~FIGS~~ 18 to 21 show another specific structure of the shift control device in accordance with another embodiment of the present invention. The structure to be described here is can be different from the above-described structure in respect to an actuation force transmission mechanism 177.

[0112] [0091]

That is, while the above-described actuation force transmission mechanism 164 drives rotationally, the actuation force transmission mechanism 177 to be described here can drives linearly. The actuation force transmission mechanism 177 is can be provided in place of the coupling rod 167 in the above-described structure. Further, it is contemplated and that the actuation force transmission mechanism 164 is may or may not be provided in the structure to be described here such an embodiment.

[0113] [0092]

As shown in the embodiments illustrated in FIGS~~FIGS~~ 18 to 21, the actuation force transmission mechanism 177 is can also be provided with first and second coupling parts 179, 180 slidably movable relative to each other in linear directions. A In such an embodiment, a coil spring 181, which is used as an “urging means,” and a stopper member 182 are can be disposed between the first and second coupling parts 179, 180.

[0114] [0093]

As shown in the embodiment of FIG. 21, the first coupling part 179 can

includes a base part 179a, and a pair of plate parts 179b, which can be fixed to the base part 179a with a constant interval. The In accordance with an implementation of such an embodiment, the two plate parts 179b are can be formed with an opening 179c where the coil spring 181 and the stopper member 182 are disposed. Further, and and the two plate parts 179b can also include with a coming-off prevention piece 179d for preventing the coil spring 181 and the stopper member 182 from coming off.

[0115] ————— [0094]

Also as shown in FIG. 21, the second coupling part 180 can includes a base part 180a, and a single plate part 180b fixed to the base part 180a. The single plate part 180b can be inserted between the pair of plate parts 179b of the first coupling part 179. The plate part 180b is can also be formed with an opening 180c generally of the same size as the opening 179c of the plate parts 179b of the first coupling part 179.

[0116] ————— [0095]

The coil spring 181 is can be accommodated in the openings 179c, 180c of the respective plate parts 179b, 180b. Further, and the columnar stopper member 182 is can be disposed inside the coil spring 181. A support shaft 183 is can be slidably inserted through the stopper member 182, and disposed between the plate parts 179b.

[0117] ————— [0096]

With this structure, to shift down, for example, the shift actuator 165 is can be driven to move the first and second coupling parts 179, 180 of the actuation force transmission mechanism 177 in compressing directions. The, and the coil spring 181 is can be compressed against its urging force from the state shown in FIG. 18 to the state shown in FIG. 19. This urging force can rotates the shift shaft 159 to allow engagement or disengagement of the dog.

[0118] ————— [0097]

When the dog is to be engaged, there are cases where the dog contacts another dog due to bad timing and hence is not engaged immediately. Even in such cases, the dogs are can be subjected to comparatively small urging force of the coil spring 181 and hence do may not abut against each other with large force. Thus, the components are can be protected from damage or the like. After that, the slide gears can move rotationally slightly,

and the urging force of the rotational movement can causes the dogs to be meshed with each other reliably.

[0119] ————— [0098]

—————As the coil spring 181 is elastically deformed and compressed, the opening 179c of the plate parts 179b and the opening 180c of the plate part 180b are-can be displaced from each other. At the time when the first and second coupling parts 179, 180 have moved relatively by a predetermined amount in linear directions, the width of an opening common to the displaced openings 179c, 180c can becomes coincident with the width of the stopper member 182. This can stops the relative movement of the first and second coupling parts 179, 180, and causes the first and second coupling parts 179, 180 to move rotationally together. Thus, even when the dog is engaged and difficult to be disengaged due to residual torque, the dog can be compulsorily disengaged.

[0120] ————— [0099]

—————On the other hand, to shift up, for example, the shift actuator 165 is-can be driven to relatively move the first and second coupling parts 179, 180 in separating directions. —Then, the opening 179c of the plate parts 179b and the opening 180c of the plate part 180b are-can be displaced from the generally coincident position, and the coil spring 181 is-can be compressed. The urging force of the coil spring 181 can tend to ensures engagement of the dog as described above.

[0121] ————— [0100]

—————Further from this state, as the coil spring 181 is elastically deformed, the opening 179c of the plate parts 179b and the opening 180c of the plate part 180b are-can be displaced from each other. At the time when the first and second coupling parts 179, 180 have moved relatively by a predetermined amount in separating directions, the width of an opening common to the displaced openings 179c, 180c can becomes coincident with the width of the stopper member 182. This can stops the relative movement of the first and second coupling parts 179, 180, and causes the first and second coupling parts 179, 180 to move rotationally together. Thus, even when the dog is engaged and difficult to be disengaged due to residual torque, the dog can be compulsorily disengaged.

[0122] ————— [0101]

— It is contemplated that the first coupling part 179, the second coupling part 180, and the stopper member 182 can be formed ~~of~~in various structures ~~are~~ conceivable configurations. Some examples exemplary embodiments are shown in FIGsFIGS. 22, 23(a) and 23(b).

[0123] — [0102]

— In the example shown in FIG. 22, the second coupling part 180 is can be constituted of a rod, and the first coupling part 179 is can be constituted of a cylindrical member for accommodating a part of the rod. The coil spring 181, utilizable as an urging means, can be is disposed between the first coupling part (cylindrical member) 179 (shown as a cylindrical member) and the second coupling part (rod) 180 (shown as a rod). A sidewall 182a inside the first coupling part 179 and a step 182b can be provided on the inner surface of the first coupling part 179 to respectively serve as stopper members when the second coupling part 180 moves relative to the first coupling part 179.

[0124] — [0103]

— For example, when the second coupling part 180 moves relative to the first coupling part 179 toward the right side of FIG. 4622, the coil spring 181 is can be compressed by a circlip 190b embedded in a portion of the first coupling part 179. The second coupling part 180 can move relatively until its distal end contacts the sidewall (stop member) 182a (utilizable as a stopper member) inside the first coupling part 179.

[0125] — [0104]

— Also, when the second coupling part 180 moves relative to the first coupling part 179 toward the left side of FIG. 4622, the coil spring 181 is can be compressed by a circlip 190a embedded in a portion of the first coupling part 179. The second coupling part 180 can move relatively until the circlip 190b embedded in a portion of the first coupling part 179 contacts the step (stopper member) 182b provided on the inner surface of the first coupling part 179.

[0126] — [0105]

— The rod and the cylindrical member constituting the first coupling part 179 and the second coupling part 180 may can be of a circular, rectangular or any other shape as long as the cylindrical member can accommodate the rod. The rod may can have portions of

different diameters, and a portion of a large diameter may be used as a part contacted by the spring.

[0127] ————— [0106]

———— In addition, the cylindrical member ~~may~~can be constituted with plural members having inner and outer surfaces. For example, the cylindrical member ~~may~~can be constituted with plural semi-cylindrical members divided along the linear direction of the rod. In this case, the cylindrical member includes plural cylindrical members.

[0128] ————— [0107]

———— ~~As illustrated in~~ In the example shown in FIG. 23(a), the distal end of the first coupling part 179 ~~is~~can be bent back and inserted into an opening of the second coupling part 180. Sidewalls 182a, 182b of the opening ~~are~~can be used as stopper members. In the example shown in FIG. 1723(b), a coil spring 181 is provided in an opening defined by the first coupling part 179 and the second coupling part 180. A projection 182a formed on the first coupling part 179 and a recess 182b formed in the second coupling part 180 ~~are~~can be fitted to each other to serve as stopper members.

[0129] ————— [0108]

———— Although the present invention has been described above by way of preferred embodiments, the above descriptions should not be construed as limitations, but various modifications may be made.

———— [0109]

———— The shift control device in embodiments of the present invention ~~may~~can be mounted on a two-wheeled motor vehicle, as shown in FIG. 6, in order to allow smooth shift change when the two-wheeled motor vehicle is running.

[0130] ————— [0110]

———— The term “two-wheeled motor vehicle” used herein ~~can~~ means include motorcycles including such as motorized bicycles (motorbikes) and scooters, and refers specifically to vehicles ~~which~~ can be whose turned turning can include by tilting of the vehicle body. Thus, a vehicle having two or more front wheels and/or two or more rear wheels and hence having a total of at least three or four (or more) wheels, can also be included in the “two-wheeled motor vehicle”. The embodiments of the present invention is

are not limited to use in two-wheeled motor vehicles, but may also be applied to other vehicles which can take advantage of the effect of embodiments of the present invention. Examples of such vehicles include the so-called straddle-type vehicles other than two-wheeled motor vehicles, such as four-wheeled buggies (all terrain vehicles (ATVs)) and snowmobiles.

[0131] ————— [0111]

————— Further, the "shift actuator" may can be of an electric or hydraulic type. Instead of pine-needle-like spring or coil spring, the "urging means" biassing mechanism may can be include another type of spring, or an elastic member, such as rubber and resin.

[0132] ————— [0112]

————— When embodiments of the present invention is are to be applied to actual straddle-type vehicles, specific implementations should be examined from a comprehensive viewpoint which allows for each and every requirement in order to produce an excellent effect such as described above.

[0133]

Industrial Applicability

————— [0113]

————— An object of the present invention is to provide Further, such implementations preferably facilitate easy installation and maintenance of embodiments of the an easy to maintain shift control device which can be used for a straddle-type vehicle, utilizing an existing structure.

[0134] Although the embodiments of the present invention have been disclosed in the context of certain preferred embodiments and examples, it will be understood by those skilled in the art that the teachings herein extend beyond the specifically disclosed embodiments to other alternative embodiments and or uses of the embodiments of the present invention and obvious modifications and equivalents thereof. In addition, while several variations of the embodiments have been shown and described in detail, other modifications, which are within the scope of these embodiments, will be readily apparent to those of skill in the art based upon this disclosure. It is also contemplated that various combination or sub-

combinations of the specific features and aspects of the embodiments may be made and still fall within the scope of the teachings. It should be understood that various features and aspects of the disclosed embodiments can be combined with or substituted for one another in order to form varying modes of the disclosed embodiments. Thus, it is intended that the scope of at least some of the embodiments herein disclosed should not be limited by the particular disclosed embodiments described above.

WHAT IS CLAIMED IS:

Claims

_____ [1] A shift control device for a straddle type vehicle for performing shift control in which a shift actuator is stroked by a predetermined amount to rotate a shift shaft, and a dog is engaged and disengaged by the rotation of the shift shaft, the device comprising:

_____ a transmission mechanism including:

_____ a first coupling part and a second coupling parts being sized and configured to be coupled together to provide for movement relative to each other in a linear direction;

_____ an urging means for urging the first and second coupling parts toward a neutral position; and

_____ a stopper mechanism for stopping the relative movement of the first or and second coupling part when one of the first or and second coupling parts is moved relatively from the neutral position against urging force of the urging means,

_____ wherin the transmission mechanism is disposed outside an engine case and interposed between the shift actuator and the shift shaft.

_____ [2] The shift control device for a straddle type vehicle according to Claim 1,

_____ wherein the transmission mechanism is arranged such that, when a resistive force acts linearly against the movement of the transmission mechanism;

_____ the first or second coupling part moves relatively to the second coupling part against the urging force of the urging means until the first or second coupling part is stopped by the stopper mechanism; and

_____ then and wherein in response to a continuing resistive force, the first and second coupling parts moveing together upon the first coupling part being stopped by the stopper mechanism.

_____ [3] The shift control device for a straddle type vehicle according to Claim 1, wherein the first and second coupling parts are coupled for sliding movement relative to each other in sliding directions.

_____ [4] The shift control device for a straddle type vehicle according to Claim 3, wherein the urging means includes a compression spring.

_____ [5] The shift control device for a straddle type vehicle according to Claim 1, wherein the first and second coupling parts are coupled for rotational movement relative to each other in rotating directions.

_____ [6] The shift control device for a straddle type vehicle according to Claim 5, wherein the urging means includes a pine needle like spring.

_____ [7] The shift control device for a straddle type vehicle according to Claim 5, wherein the transmission mechanism is disposed on the shift shaft.

_____ [8] The shift control device for a straddle type vehicle according to Claim 7, wherein the transmission mechanism is disposed on a gear shaft of a speed reduction mechanism coupled to the shift actuator.

_____ [9] The shift control device for a straddle type vehicle according to Claim 1, wherein:

_____ the shift actuator is coupled to the shift shaft via a coupling mechanism for transmitting actuation force of the shift actuator to the shift shaft; and

_____ the transmission mechanism is being held by the coupling mechanism.

_____ [10] The shift control device for a straddle type vehicle according to Claim 9, wherein the transmission mechanism is provided in a case held by the coupling mechanism.

[11] The shift control device for a straddle type vehicle according to
Claim 1, wherein:

the shift actuator is coupled to the shift shaft via a coupling mechanism
for transmitting actuation force of the shift actuator; and

the coupling mechanism is being of adjustable length.

[12] A straddle type vehicle incorporating the shift control device
according to any one of Claims 1, 9, 10 and 11.

SHIFT CONTROL DEVICE FOR STRADDLE-TYPE VEHICLE, AND STRADDLE-TYPE VEHICLE

ABSTRACT OF THE DISCLOSURE

Abstract

[Abstract]

[Problem to be Solved] To provide an easy to maintain shift control device for a straddle type vehicle utilizing an existing structure.

[Solution] A transmission mechanism 10 is provide that can be interposed between a shift actuator and a shift shaft. The mechanism can 10 includes: a first coupling part 11a and a second coupling part 11b that can be coupled for movement relative to each other; an urging means biasing mechanism that can be configured 12 for urging the first and second coupling parts 11a, 11b toward a neutral position; and a stopper mechanism 13 that can be configured for stopping the relative movement of one of the first or and second coupling parts when the first or second coupling part is moved relatively from the neutral position against urging force of the urging means biasing mechanism. When the shift actuator is stroked by a predetermined amount, a dog is can be compulsorily disengaged as the first and second coupling parts 11a, 11b are moved together by means of the stopper mechanism 13, and engaged as one of the first or and second coupling parts 11a, 11b is moved relatively against the urging force, for allowing of the urging means 12. This allows smooth shift change.

[Selected Drawing] FIG.1

2838073
081506
2883246
082906